

# WORMS AND WORM GEARS

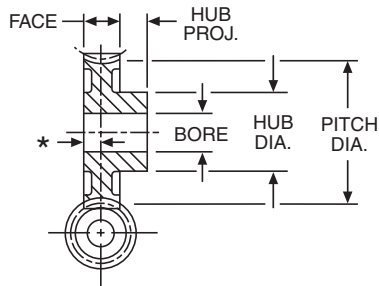
## 12 DIAMETRAL PITCH

### BRONZE AND CAST IRON WORM GEARS

### STEEL WORMS – UNHARDENED AND HARDENED

PRESSURE ANGLE – 14½°

RATIO = Gear Teeth ÷ Worm Threads  
RH = RIGHT HAND — LH = LEFT HAND  
All others stocked RIGHT HAND ONLY.

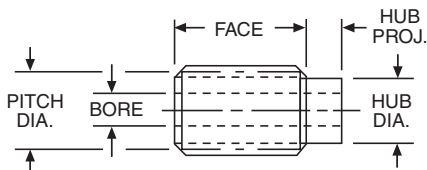


#### STANDARD TOLERANCES

DIMENSION		TOLERANCE
BORE	All	±.0005

#### WORM LEAD and LEAD ANGLE

	SINGLE	DOUBLE	QUAD
LEAD	.2618"	.5236"	1.0472"
LEAD ANGLE	4°46'	9°28'	18°26'



#### REFERENCE PAGES

Alterations — 149  
Horsepower Ratings — 85, 86  
Lubrication — 149  
Materials — 150  
Selection Procedure — 84

ALL DIMENSIONS IN INCHES  
ORDER BY CATALOG NUMBER OR ITEM CODE

12 DIAMETRAL PITCH		WORM GEARS				FACE = .500" *CENTER LINE WORM TO FLUSH END = .250"						
No. of Teeth	Pitch Dia.	Bore	Hub		Style See Page 150	SINGLE Thread		DOUBLE Thread		QUAD Thread		
			Dia.	Proj.		Catalog Number	Item Code	Catalog Number	Item Code	Catalog Number	Item Code	
<b>BRONZE</b>												
12	1.000	.5625	1.00	.62	A	—	—	—	—	QB1212†	13762	
16	1.333	.6875	1.25	.62		—	—	—	—	QB1216†	13764	
20	1.667	.500	1.25	.50		GB1050A	13626	DB1400	13714	DB1600	13766	
30	2.500	.500	1.19	.62	B	GB1051	13628	—	—	—	—	
		.750		.62		—	—	DB1401A	13716	—	—	—
40	3.333	.625	1.44	.62		GB1052A	13630	—	—	—	—	—
		.750		.62		—	—	DB1402A	13718	—	—	—
50	4.167	.625	1.44	.62		GB1053A	13632	—	—	—	—	—
		.750		.62		—	—	DB1403A	13720	—	—	—
60	5.000	.625	1.69	.62	C	GB1260A	13634	—	—	—	—	
		.750		.62		—	—	DB1260A	13722	—	—	—
80	6.667	.625	1.94	.75		GB1054	13636	—	—	—	—	—
100	8.333	.750	1.94	.75	GB1055	13638	—	—	—	—	—	
<b>CAST IRON</b>												
20	1.667	.500	1.25	.50	A	G1050ARH	13110	D1400RH	13260	D1600	13352	
				.62		G1050ALH	13112	D1400LH	13262	—	—	—
30	2.500	.500	1.19	.62		G1051RH	13114	—	—	—	—	—
		.750		.62	G1051LH	13116	—	—	—	—	—	
40	3.333	.625	1.44	.62	B	G1052ARH	13118	D1401ARH	13264	D1601A	13354	
		.750		.62		G1052ALH	13120	D1401ALH	13266	—	—	—
				.62		—	—	D1402ARH	13268	—	—	—
				.62		—	—	D1402ALH	13270	D1602A	13356	
50	4.167	.625	1.44	.62		G1053ARH	13122	—	—	—	—	—
		.750		.62		G1053ALH	13124	—	—	—	—	—
60	5.000	.625	1.69	.75	C	G1260RH	13126	D1403ARH	13272	D1603A	13358	
		.750		.75		G1260LH	13128	D1403ALH	13274	—	—	—
				.75		—	—	D1260A	13276	—	—	—
80	6.667	.625	1.94	.75		G1054	13130	—	—	—	—	—
		.750		.75	—	—	D1404	13278	—	—	—	
100	8.333	.750	1.94	.75	G1055	13134	—	—	—	—	—	

12 DIAMETRAL PITCH		WORMS FOR ABOVE GEARS									
Pitch Dia.	Face	Bore	Hub		SINGLE Thread		DOUBLE Thread		QUAD Thread		
			Dia.	Proj.	Catalog Number	Item Code	Catalog Number	Item Code	Catalog Number	Item Code	
<b>UNHARDENED – STEEL</b>											
	1.125	.625	—	—	—	—	D1407KRH‡	12806	D1607KRH‡	12822	
		.625	—	—	—	—	D1407KLH‡	12808	D1607KLH‡	12824	
1.000	1.625	.500	—	—	L1056‡	12900	L1407‡	12912	—	—	
	1.125	.500	.75	.38	GH1056RH	12884	DH1407RH	12838	DH1607	12854	
					GH1056LH	12886	DH1407LH	12840	—	—	
<b>HARDENED – STEEL</b>											
	1.125	.625	—	—	—	—	H1407RH‡	12980	H1607‡	12996	
		.625	—	—	—	—	H1407LH‡	12982	—	—	
1.000	1.625	.500	—	—	HL1056‡	13006	—	—	—	—	
	1.125	.500	—	—	H1056RH‡	12962	—	—	—	—	
					H1056LH‡	12960	—	—	—	—	

‡.750" Face, Center Line Worm to Flush End = .375"  
‡Furnished with .125" Keyway.  
Hardened Worms have ground and polished threads.

# ENGINEERING INFORMATION

## GENERAL

### MATERIALS

Boston Gear stock steel gears are made from a .20 carbon steel with no subsequent treatment. For those applications requiring increased wearability. Case-hardening produces a wear resistant, durable surface and a higher strength core. Carburizing and hardening is the most common process used. Several proprietary nitriding processes are available for producing an essentially distortion-free part with a relatively shallow but wear-resistant case. Boston stock worms are made of either a .20 or .45 carbon steel. Selection of material is based on size and whether furnished as hardened or untreated.

Stock cast iron gears are manufactured from ASTM-CLASS 30 cast iron to Boston Gear specifications. This provides a fine-grained material with good wear-resistant properties.

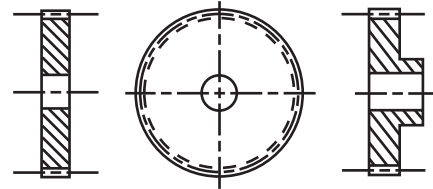
Bronze worm and helical gears are produced from several alloys selected for bearing and strength properties. Phosphor bronze is used for helicals and some worm gears (12P and coarser). Finer pitch worm gears are made from several different grades of bronze, dependent on size.

Non-metallic spur Gears listed in this Catalog are made from cotton reinforced phenolic normally referred to as Grade "C."

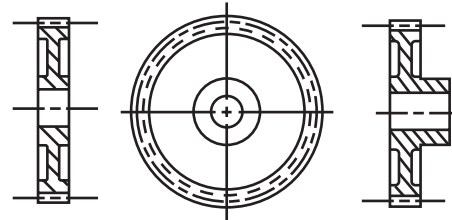
Plastic Gears listed are molded from either Delrin®, Acetal or Minlon®.

### STYLES

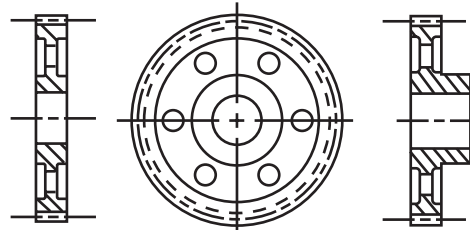
Boston Spur, Helical, and Worm Gears are carried in Plain, Web, or Spoke styles, as illustrated.



PLAIN – A



WEB – B



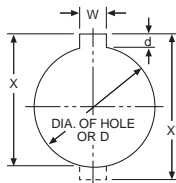
WEB WITH LIGHTNING HOLES – C



SPOKE – D

### STANDARD KEYWAYS AND SETSCREWS

Diameter of Hole	Standard		Recommended Setscrew
	W	d	
5/16 to 7/16"	3/32"	3/64"	10-32
1/2 to 9/16	1/8	1/16	1/4-20
5/8 to 7/8	3/16	3/32	5/16-18
15/16 to 1-1/4	1/4	1/8	3/8-16
1-5/16 to 1-3/8	5/16	5/32	7/16-14
1-7/16 to 1-3/4	3/8	3/16	1/2-13
1-13/16 to 2-1/4	1/2	1/4	9/16-12
2-5/16 to 2-3/4	5/8	5/16	5/8-11
2-13/16 to 3-1/4	3/4	3/8	3/4-10
3-5/16 to 3-3/4	7/8	7/16	7/8-9
3-13/16 to 4-1/2	1	1/2	1-8
4-9/16 to 5-1/2	1-1/4	7/16	1-1/8-7
5-9/16 to 6-1/2	1-1/2	1/2	1-1/4-6



### FORMULA:

$$X = \sqrt{(D/2)^2 - (W/2)^2} + d + D/2$$

$$X' = 2X - D$$

### EXAMPLE:

Hole 1"; Keyway 1/4" wide by 1/8" deep.

$$X = \sqrt{(1/2)^2 - (1/8)^2} + 1/8 + 1/2 = 1.109"$$

$$X' = 2.218 - 1.000 = 1.218"$$

## BOSTON GEAR®

# ENGINEERING INFORMATION

## HOW TO FIGURE HORSEPOWER AND TORQUE

TO OBTAIN	HAVING	FORMULA
Velocity (V) Feet Per Minute	Pitch Diameter (D) of Gear or Sprocket – Inches & Rev. Per Min. (RPM)	$V = .2618 \times D \times \text{RPM}$
Rev. Per Min. (RPM)	Velocity (V) Ft. Per Min. & Pitch Diameter (D) of Gear or Sprocket—Inches	$\text{RPM} = \frac{V}{.2618 \times D}$
Pitch Diameter (D) of Gear or Sprocket — Inches	Velocity (V) Ft. Per Min. & Rev. Per Min. (RPM)	$D = \frac{V}{.2618 \times \text{RPM}}$
Torque (T) In. Lbs.	Force (W) Lbs. & Radius (R) Inches	$T = W \times R$
Horsepower (HP)	Force (W) Lbs. & Velocity (V) Ft. Per Min.	$\text{HP} = \frac{W \times V}{33000}$
Horsepower (HP)	Torque (T) In. Lbs. & Rev. Per Min. (RPM)	$\text{HP} = \frac{T \times \text{RPM}}{63025}$
Torque (T) In. Lbs.	Horsepower (HP) & Rev. Per Min. (RPM)	$T = \frac{63025 \times \text{HP}}{\text{RPM}}$
Force (W) Lbs.	Horsepower (HP) & Velocity (V) Ft. Per Min.	$W = \frac{33000 \times \text{HP}}{V}$
Rev. Per Min. (RPM)	Horsepower (HP) & Torque (T) In. Lbs.	$\text{RPM} = \frac{63025 \times \text{HP}}{T}$

**POWER** is the rate of doing work.

**WORK** is the exerting of a **FORCE** through a **DISTANCE**. **ONE FOOT POUND** is a unit of **WORK**. It is the **WORK** done in exerting a **FORCE OF ONE POUND** through a **DISTANCE** of **ONE FOOT**.

**THE AMOUNT OF WORK** done (Foot Pounds) is the **FORCE** (Pounds) exerted multiplied by the **DISTANCE** (Feet) through which the **FORCE** acts.

**THE AMOUNT OF POWER** used (Foot Pounds per Minute) is the **WORK** (Foot Pounds) done divided by the **TIME** (Minutes) required.

$$\text{POWER (Foot Pounds per Minute)} = \frac{\text{WORK (Ft. Lbs.)}}{\text{TIME (Minutes)}}$$

**POWER** is usually expressed in terms of **HORSEPOWER**.

**HORSEPOWER** is **POWER** (Foot Pounds per Minute) divided by 33000.

$$\begin{aligned} \text{HORSEPOWER (HP)} &= \frac{\text{POWER (Ft. Lbs. per Minute)}}{33000} \\ &= \frac{\text{WORK (Ft. Pounds)}}{33000 \times \text{TIME (Min.)}} \\ &= \frac{\text{FORCE (Lbs.)} \times \text{DISTANCE (Feet)}}{33000 \times \text{TIME (Min.)}} \\ &= \frac{\text{FORCE (Lbs.)} \times \text{DISTANCE (Feet)}}{33000 \times \text{TIME (Min.)}} \end{aligned}$$

Cut on Dotted Lines  
and Keep for Quick Reference

### APPLICATION FORMULAS

$$\begin{aligned} 1 \text{ hp} &= 36 \text{ lb-in. @ 1750 rpm} \\ 1 \text{ hp} &= 3 \text{ lb-ft. @ 1750 rpm} \end{aligned}$$

$$\text{hp} = \frac{\text{Torque (lb.-in.)} \times \text{rpm}}{63,025}$$

$$\text{hp} = \frac{\text{Force (lb.)} \times \text{Velocity (ft./min.)}}{33,000}$$

$$\begin{aligned} \text{Velocity (ft./min.)} &= 0.262 \times \text{Dia. (in.)} \times \text{rpm} \\ \text{Torque (lb.-in.)} &= \text{Force (lb.)} \times \text{Radius (in.)} \end{aligned}$$

$$\text{Torque (lb.-in.)} = \frac{\text{hp} \times 63,025}{\text{rpm}}$$

$$\text{Mechanical Efficiency} = \frac{\text{Output hp}}{\text{Input hp}} \times 100\%$$

$$\text{Output hp} = \frac{\text{OT (lb.-in.)} \times \text{Output rpm}}{63,025}$$

$$\begin{aligned} \text{OT} &= \text{Input Torque} \times \text{Ratio} \times \text{Efficiency} \\ \text{OT} &= \text{Output Torque} \end{aligned}$$

$$\text{Output rpm} = \frac{\text{Input rpm}}{\text{Ratio}}$$

$$\text{OHL} = \frac{2TK}{D}$$

$$\begin{aligned} \text{OHL} &= \text{Overhung Load (lb)} \\ T &= \text{Shaft Torque (lb.-in.)} \\ D &= \text{PD of Sprocket, Pinion or Pulley (in.)} \\ K &= \text{Overhung Load Factor} \end{aligned}$$

Overhung Load Factors:

Sprocket or Timing Belt	1.00
Pinion & Gear Drive	1.25
Pulley & V-Belt Drive	1.50
Pulley & Flat Belt Drive	2.50
Variable Pitch Pulley	3.50

$$\text{kW} = \text{hp} \times 0.7457$$

$$\text{in.} = \text{mm}/25.4$$

$$\text{Temp. } ^\circ\text{C} = (^\circ\text{F} - 32) \times 0.556$$

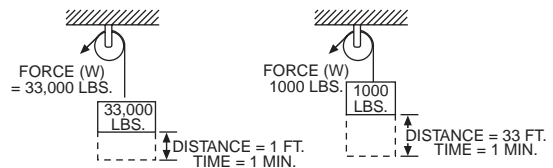
$$\text{Temp. } ^\circ\text{F} = (^\circ\text{C} \times 1.8) + 32$$

$$\text{Torque (lb.-in.)} = 86.6 \times \text{kg}\cdot\text{m}$$

$$\text{Torque (lb.-in.)} = 8.85 \times \text{N}\cdot\text{m}$$

$$\text{Torque (lb.-in.)} = 88.5 \times \text{daN}\cdot\text{m}$$

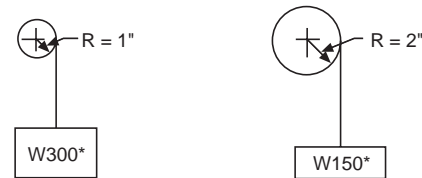
### ILLUSTRATION OF HORSEPOWER



$$\text{HP} = \frac{33,000 \times 1}{33,000 \times 1} = 1 \text{ HP}$$

$$\text{HP} = \frac{1000 \times 33}{33,000 \times 1} = 1 \text{ HP}$$

**TORQUE (T)** is the product of a **FORCE (W)** in pounds, times a **RADIUS (R)** in inches from the center of shaft (Lever Arm) and is expressed in Inch Pounds.



$$T = WR = 300 \times 1 = 300 \text{ In. Lbs.}$$

$$T = WR = 150 \times 2 = 300 \text{ In. Lbs.}$$

If the shaft is revolved, the **FORCE (W)** is moved through a distance, and **WORK** is done.

$$\text{WORK (Ft. Pounds)} = W \times \frac{2\pi R}{12} \times \text{No. of Rev. of Shaft.}$$

When this **WORK** is done in a specified **TIME**, **POWER** is used.

$$\text{POWER (Ft. Pounds per Min.)} = W \times \frac{2\pi R}{12} \times \text{RPM}$$

Since (1) **HORSEPOWER** = 33,000 Foot Pounds per Minute

$$\text{HORSEPOWER (HP)} = W \times \frac{2\pi R}{12} \times \frac{\text{RPM}}{33,000} = \frac{W \times R \times \text{RPM}}{63,025}$$

but **TORQUE (Inch Pounds)** = **FORCE (W)** X **RADIUS (R)**

$$\text{Therefore HORSEPOWER (HP)} = \frac{\text{TORQUE (T)} \times \text{RPM}}{63,025}$$